

**CSTB Workshop:
Matlab/Simulink Building and HVAC
Simulation
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**Modelling Small-scale CHP Plant Under Closed
Loop Control**

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Background – UK Climate Change Levy

- A commodities tax introduced in April 2001
- Applicable to electricity, gas and other fuels (but not oil since this is already subject to excise duty)
- Payable by most (there are a few exceptions) non-domestic energy consumers
- Examples of current Levy rates: 0.15 p/kWh (natural gas); 0.43 p/kWh (electricity)



Background – CHP and the Levy

- Purpose of the Levy: to help secure the UK's CO₂ emissions reduction target (5% (Kyoto-binding); 12.5% targeted by between 2008 – 2012)
- Fuels used in “good quality” CHP will be exempt from the Levy
- UK CHP capacity target of 10 GWe by 2010 (about 15% of current capacity)



CHP Quality Indexing

- Quality indexing scheme operated by the CHPQA group at the UK's DEFRA
- *QI* thresholds set for different types of CHP installation
- A *QI* is calculated as a weighting of power and heat from CHP that is actually utilised
- Example, for small scale installations < 1 MWe...

Threshold: $\eta \geq 0.2$ AND $QI \geq 115$

Where: $QI = 230\eta + 125\phi$

(η = efficiency; ϕ = heat "efficiency")

Modelling Equations – Gas and Work

$$\eta_{\text{indicated}} = 1 - r_c^{1-\gamma}$$

$$\dot{W}_{\text{indicated}} = \dot{V} \cdot V_{\text{disp}}$$

$$P_3 = \text{fnc}(v_{\text{cyl}}, T_{\text{ai}}, M_f, \eta_{\text{indicated}}, FLHV, W_{\text{brake}})$$

$$P_2 = \max(P_3, P_{\text{crit}})$$

$$M_g = \text{fnc}(A_{\text{throat}}, v_{\text{ai}}, T_{\text{ai}}, P_2, P_{\text{ai}})$$

$$A_{\text{throat}} = \text{fnc}(IV_{\text{lift}}, IV_{\text{diam}})$$

$$W_{\text{brake}} = \text{fnc}(M_g, \eta_{\text{indicated}}, \eta_{\text{mech}}, FLHV)$$

$$M_f = \text{fnc}(FAR, M_g)$$

(M_g is solved recursively)



Modelling Equations – Heat Transfer

$$T_{\text{adiabatic}} = \text{fnc}(h_{\text{gas-chamber}}, FLHV, FAR)$$

($T_{\text{adiabatic}}$ implicit in $h_{\text{gas-chamber}}$ hence solve recursively)

$$T_{\text{gas-outlet}} = \text{fnc}(T_{\text{adiabatic}}, W_{\text{brake}})$$

$$Q_{\text{HX}} = \text{fnc}(T_{\text{gas}}, T_{\text{coolant}}, E_{\text{HX}})$$

$$Q_{\text{loss}} = \text{fnc}(T_{\text{coolant}}, T_{\text{ambient}}, E_{\text{loss}})$$

Modelling - Uncertainties

- Inlet throat area fitted to mnfrs. data
- General form...

$$IV_{\text{lift}} = C_1 + C_2 V_{\text{disp}} + \dots$$

$$C_3 V_{\text{disp}}^2 + C_4 r_c + \dots$$

$$C_5 r_c^2 + C_6 r_c V_{\text{disp}} + \dots$$

$$C_7 (r_c V_{\text{disp}})^2$$

- Mechanical efficiency balanced from mnfrs. data
- Generally lower for smaller engines
- Typically, for naturally aspirated SI engines up to 250 kW...

$$0.6 \leq \eta_{\text{mech}} \leq 0.82$$

Matlab Function - ChpSim

PARAMETER SET

Number of Cylinders:

Cylinder Bore (mm):

Displacement Volume (all cylinders - litres):

Compression Ratio (typically: 10 (SI), 20 (CI)):

Mechanical Efficiency (fraction):

INPUT DATA SET

Relative Air:Fuel Ratio (Stoichiometric=1):

Jacket Water Mass Flow rate (kg/s):


Jacket Inlet Water Temperature (K):

Inlet Air Pressure (N/sq m):

Inlet Air Temperature (K):

Engine Speed (RPS):

Jacket-series, separate circuit or...

 Auxiliary cooling

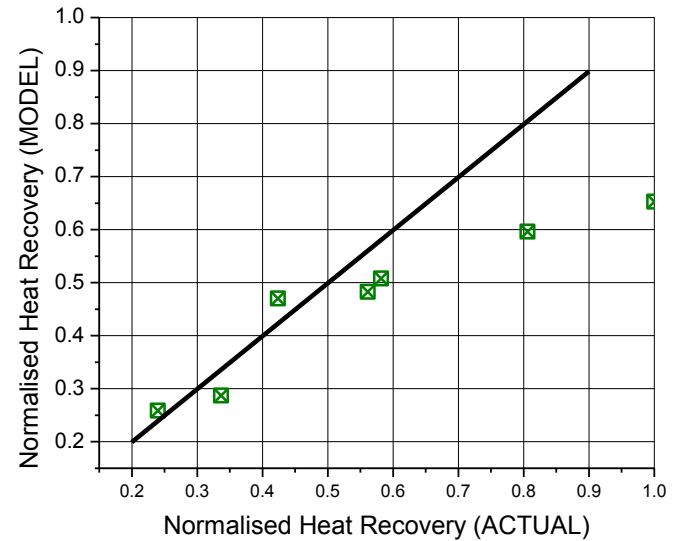
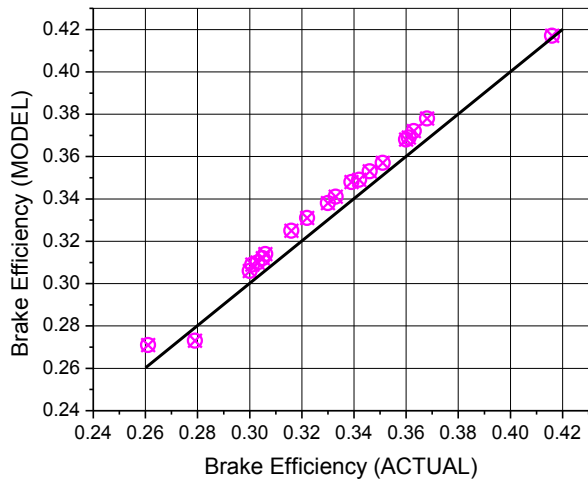
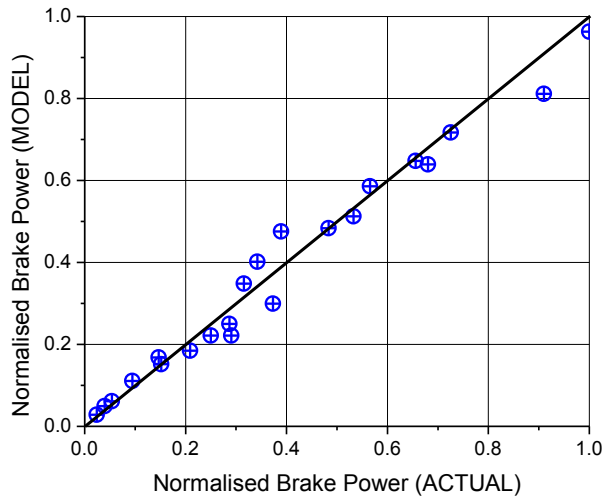
AUXILLARY HX SIZE

Aux. Heat Exchanger Surface Area (sq m):

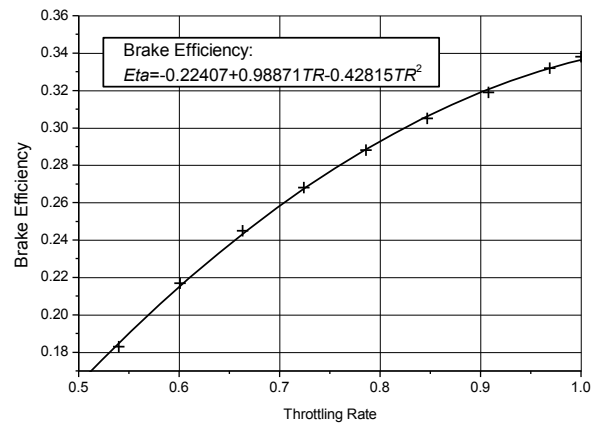
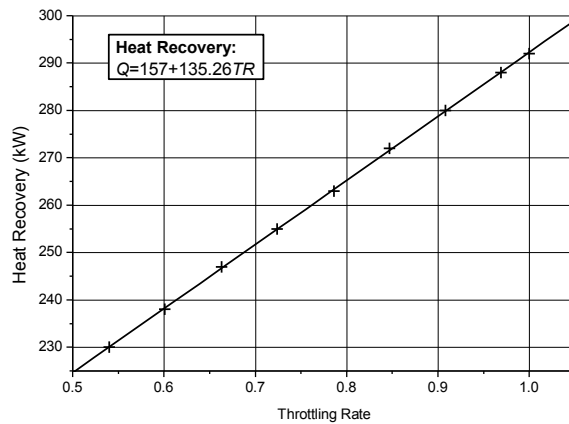
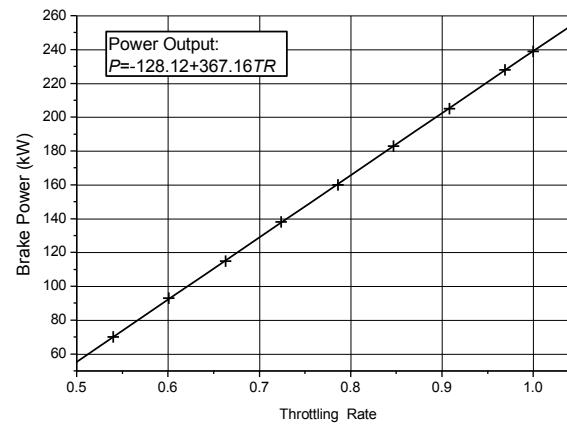
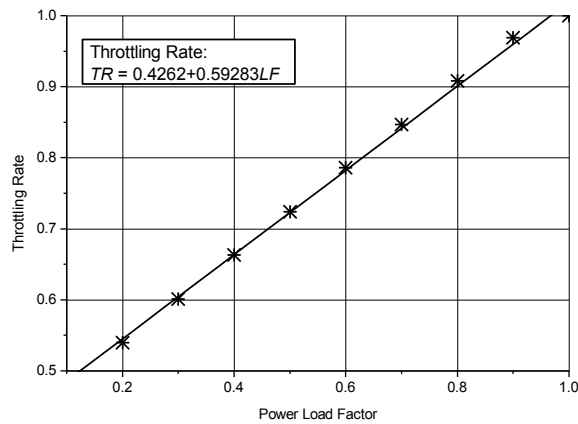
Manufacturers Reference Data

Manufacturer	Model	Brake Power (kW)	Fuel Use (kW)	Brake Efficiency	Displacement (Litre)	Compression Ratio
MAN Rollo	E0824E302	37	142	0.261	4.58	-
	E0826E302	70	210	0.333	6.873	-
	E2866E	95	271	0.351	11.97	-
	E2866E302	118	341	0.346	11.97	-
	E2876E302	130	376	0.346	12.82	-
	E2842E	177	490	0.361	21.94	-
	E2842E302	222	617	0.360	21.94	-
Perkins	3008SI	160	445	0.360	17.41	12
Waukesha	F11G	83.5	273	0.306	11.03	10
Caterpillar	G3304	61	185	0.330	6.994	10.5
	G3304	51	167	0.305	6.994	8
	G3306	91	266	0.342	10.49	10.5
	G3306	77	257	0.300	10.49	8
	G3406	138	407	0.339	14.6	10.3
	G3408	166	551	0.301	17.93	10
	G3412	244	758	0.322	26.9	10
Cummins	Onan LPG-2	5.8	20.8	0.279	0.928	9.5
	Onan LPG-3	9.7	30.7	0.316	1.391	9.5
	Onan LPG-4	13.2	43.6	0.303	1.855	9.5
	Ford LRG-4251	23.1	55.5	0.416	2.451	9.4
	Ford ESG-642	35.8	97.2	0.368	4.197	9.3
	Ford WSG-1068	70.9	195.3	0.363	6.77	^

Correlation with Manufacturers Data

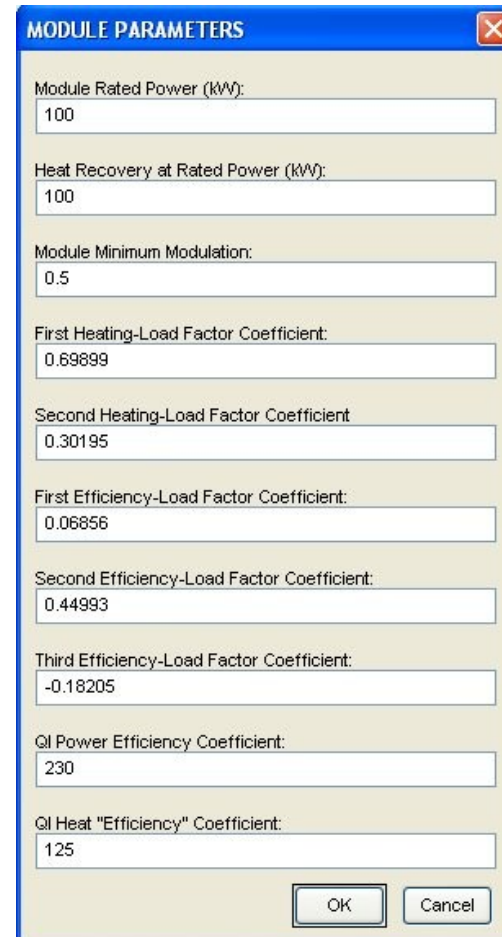


Parameter Extraction Using ChpSim



Matlab Function ChpScheduler

- Reads hourly time series of heat & power demands
- Performs an hourly balance to give...
 - Module matching
 - Turndown
 - Energy balance
 - Machine utilisation
- Integrated annual energy balance includes QI



MODULE PARAMETERS

Module Rated Power (kW):
100

Heat Recovery at Rated Power (kW):
100

Module Minimum Modulation:
0.5

First Heating-Load Factor Coefficient:
0.69899

Second Heating-Load Factor Coefficient:
0.30195

First Efficiency-Load Factor Coefficient:
0.06856

Second Efficiency-Load Factor Coefficient:
0.44993

Third Efficiency-Load Factor Coefficient:
-0.18205

QI Power Efficiency Coefficient:
230

QI Heat "Efficiency" Coefficient:
125

OK Cancel

Application Example: 700-household Village



- “Heat-rich” case
- Two scenarios...
 - existing case
 - 50% reduced heating to reflect a major insulation campaign

Simulated Village Energy Demands

Cockfield Village: Predicted Global Domestic Energy Demands

Bold pattern: electricity demand Feint pattern: heating demand





Application – Nominal Module Choice

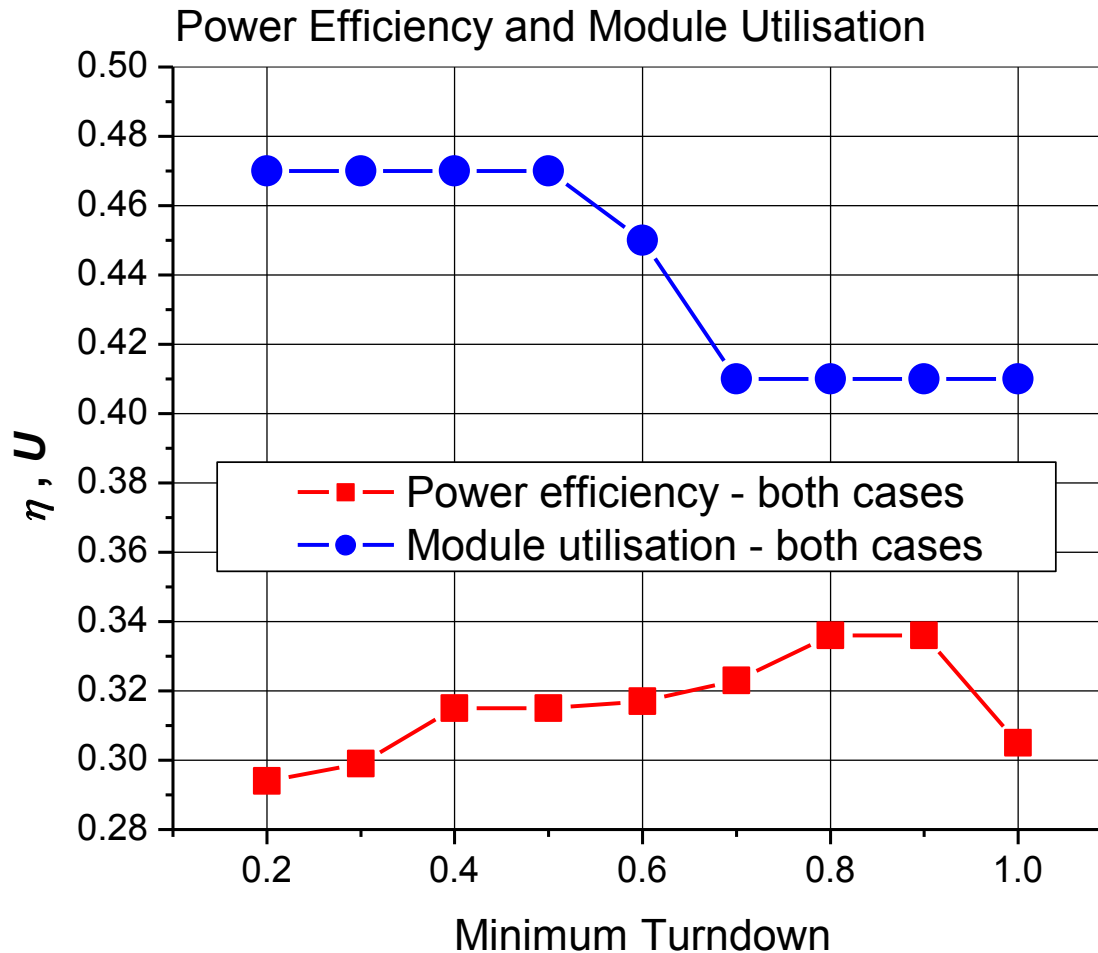
- Naturally aspirated gas engine
- 8-cylinder in-line; 137 mm bore; 26.9 ltr.; $r_c = 10$; $\eta_{\text{mech}} = 0.65$; $M_w = 5 \text{ kgs}^{-1}$ (series cooled)
- Parameter extraction from *ChpSim...*

$$Q_{\text{recovery}} = 0.699 + 0.302 (W / W_{\text{rated}})$$

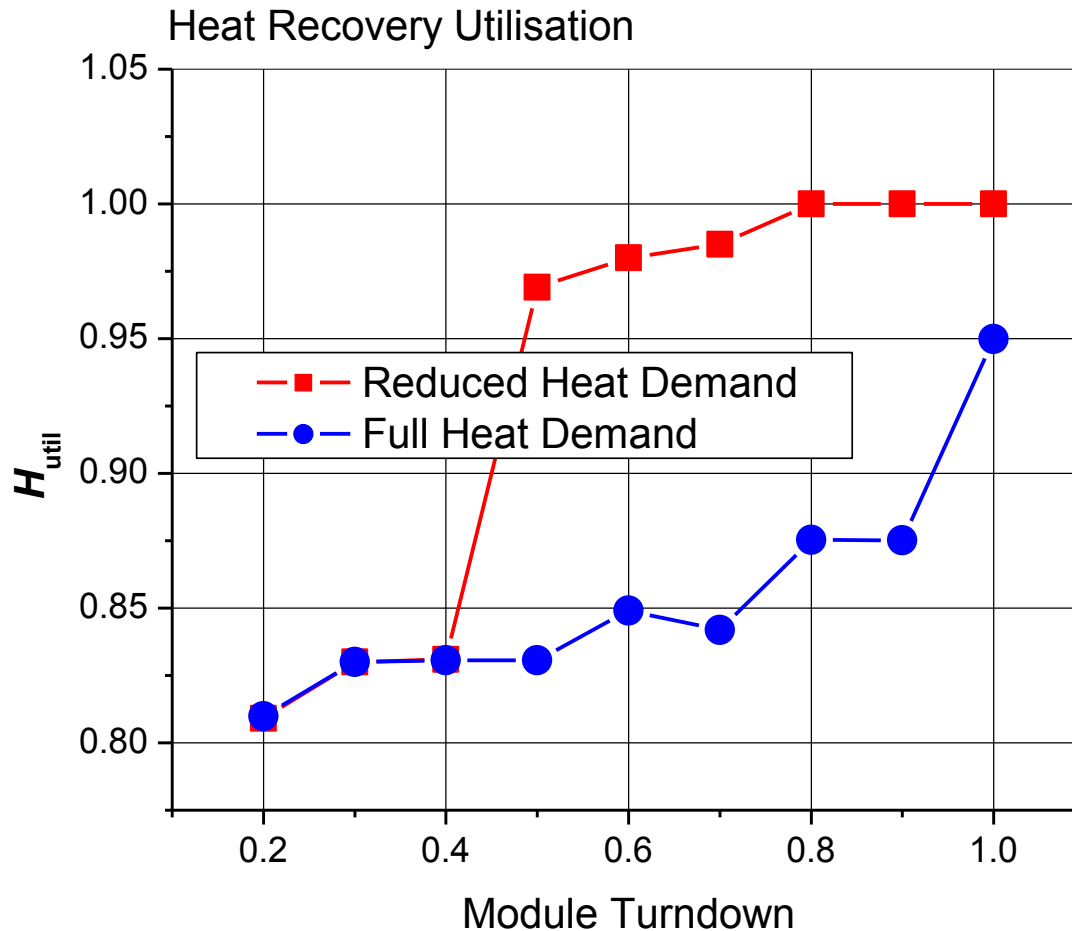
$$\eta = 0.0686 + 0.450 (W / W_{\text{rated}}) - 0.182 (W / W_{\text{rated}})^2$$

- Simulated nominal capacities 239 kWe / 292 kWt

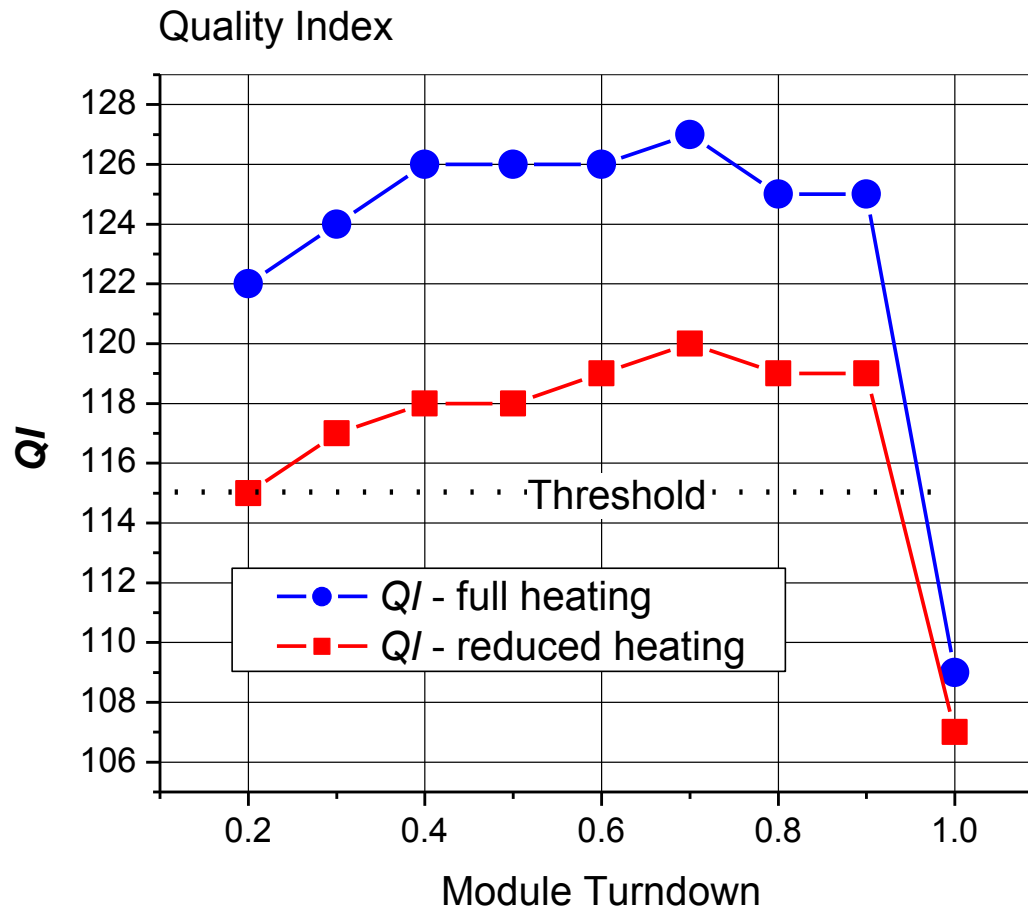
Results – Efficiency and Module Utilisation



Results – Heat Recovery Utilisation



Results – Quality Index





Conclusions and Further Work

- For maximised QI and η , a CHP module must be capable of turndown though this will reduce module utilisation
- The minimum turndown is shown here to maximise QI when set at 0.7 whereas for maximised η it should be 0.8 – 0.9
- For a “heat rich” application, heat recovery utilisation is maximised when no turndown is applied
- Further work is required to investigate short term module dynamics, smart control and thermal storage
- Further work is also needed to extend the range of model applicability to large turbo/super-charged engines and gas turbines